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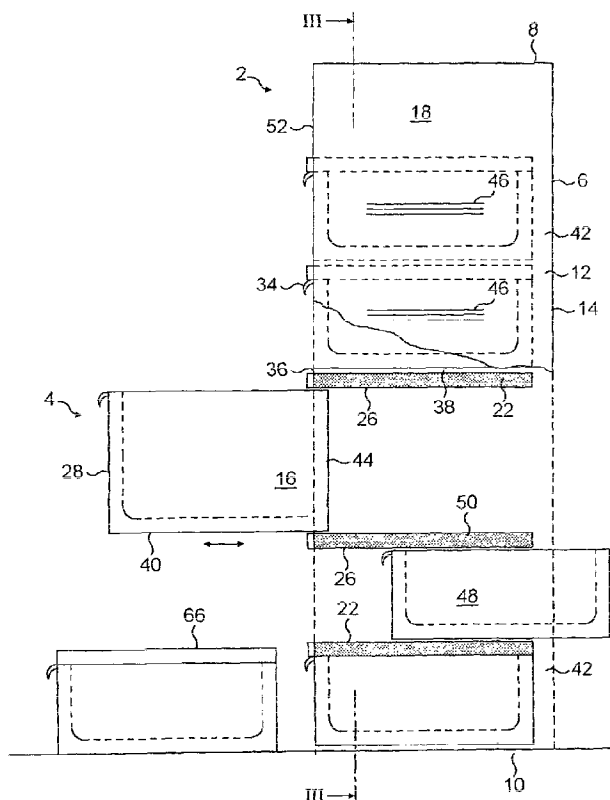
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(54) Title: AIRFLOW MANAGEMENT IN COLD STORAGE APPLIANCES



(57) Abstract: A cold-storage appliance (2) includes an open-topped insulating container (16) defining an external surface; an insulating lid (22) adapted to close the open top of the container (16); a cooling means (24) adapted to cool the interior of the container (16); and a structure supporting the container (16), the lid and the cooling means; wherein the container (16) is mounted to the structure (6) for movement relative to the structure (6) and the lid (22) to open the container (16) and afford access to its interior or to close the container (16). A cold-storage appliance with air transfer valves (352, 356) that mitigate the piston effect experienced when opening and closing a drawer (4). Bellows (374) attached to the drawer (4) to maintain the vapour barrier.



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## **AIRFLOW MANAGEMENT IN COLD STORAGE APPLIANCES**

5 This invention relates to the art of cold storage, including appliances such as refrigerators and freezers for storing foodstuffs and other perishables. Other applications of the invention include storage of chemicals and medical or biological specimens. The invention also finds use in mobile applications, for example in the transport and storage of perishable goods.

10 The invention develops and adds to the various features of the Applicant's co-pending International Patent Application No. PCT/GB00/03521 published as WO 01/20237, the content of which is incorporated herein by reference. The invention is also derived from UK Patent Application No. 0106164.7 published as GB 2367353, the content of which is also incorporated herein by reference and from which, *inter alia*, the present application claims priority. As in those specifications, the invention can be applied to storing any  
15 items within a cooled environment, such as in a refrigerated goods vehicle. The term 'appliance' is therefore to be construed broadly, extending beyond fixed domestic devices into industrial, scientific and mobile applications. However, this specification will particularly describe domestic or commercial cold-storage appliances for storing foodstuffs.

20 Briefly to recap the introduction of WO 01/20237, the advantages of storing foodstuffs and other perishable items in refrigerated and segregated conditions have long been known: refrigeration retards the degradation of such items and segregation helps to prevent their cross-contamination. Accordingly, modern cold-storage appliances such as refrigerators  
25 and freezers are usually compartmentalised, albeit not often effectively, so that a user can store different types of food in different compartments. All such appliances have the additional aim of maximising their energy efficiency.

30 The invention herein and the inventions in WO 01/20237 and GB 2367353 were devised against a background of typical cold-storage appliances, most of which comprise one or more upright cabinets each with a vertically-sealed hinged door on its front. Substantially all of the interior of the cabinet defines a storage volume, most commonly partitioned by shelves or drawers for supporting stored foodstuffs. Access to all of the shelves or drawers

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in the cabinet is gained by opening the door.

A cooler unit generates a convection loop within the cabinet, in which air cooled by the cooler unit sinks toward the bottom of the cabinet and as that air absorbs heat during its downward journey, it warms and rises back up to the cooler unit where it is cooled again. It is also possible to have forced-air circulation by means of a fan within or communicating with the cabinet. The shelves or drawers are typically made of wire so that they offer little resistance to this circulation of air.

Upright refrigerators and freezers are often combined and sold as a single-cabinet 'fridge freezer' unit with a refrigerator occupying an upper compartment and the freezer occupying a lower compartment, or vice versa. As different temperatures are required for the two compartments, they are partitioned by a solid divide and each compartment has its own door and cooler unit, conventionally in the form of an evaporator.

The domestic fridge freezer usually has only one compressor and the refrigerator evaporator is in series with the freezer evaporator. In that case, temperature control and measurement is usually confined to the refrigerator compartment. Where temperature control is required in both compartments, the evaporators are in parallel and have respective solenoid valves and temperature switches providing on/off cooling mass control to each compartment. In either case, however, the temperature within the respective compartments cannot be duplicated: one compartment is for chilling, so it has less insulation than the other and its temperature can be adjusted within a range above zero Celsius, and the other is for freezing, so it has more insulation than the other and its temperature can be adjusted (if at all) within a range below zero Celsius. Neither compartment can do the job of the other.

WO 01/20237 addresses a major problem with upright refrigerators and freezers, namely the upright door which, when opened, allows cold air to flow freely out of the cabinet to be replaced by warm ambient air flowing in at the top. That rush of ambient air into the cabinet causes its internal temperature to rise, hence consuming more energy in redressing that rise by running the cooler unit. The incoming ambient air introduces the possibility of

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airborne contamination, and moisture in that air also gives rise to condensation and ice within the cabinet. The more often and frequently the cabinet is opened, as may happen especially in commercial cold storage appliances, the worse these problems get.

5 In upright-door arrangements, the limitations of the vertical seal mean that loss of cold air and induction of warm air can even occur when the door is closed. Being denser than warmer air, the coldest air collects at the bottom of the cabinet and applies pressure to the sealing interface so that unless the seal forms a perfect seal between the door and the cabinet, that air will escape.

10

The present invention and WO 01/20237 also address the problems inherent in the well-known chest freezer, whose open-topped cabinet is typically closed by a horizontally-hinged upwardly-opening lid. Such a chest freezer is inconvenient and wasteful of space because it precludes use of the space immediately above the freezer, which space must be  
15 preserved to allow its lid to be opened. Even if a sliding lid is used instead of an upwardly-opening lid, items cannot be left conveniently on top of the lid. It is also well known that large chest freezers can make access to their contents extremely difficult, it being necessary to stoop down and shift numerous heavy and painfully cold items to get to items at the bottom of the freezer compartment.

20

Finally, the present invention and WO 01/20237 address the problem of segregating different types of foodstuff or other perishable items to avoid cross-contamination. In typical cold-storage appliances, segregation of food is compromised by the convection and/or forced-air principles on which those appliances rely. The substantially open baskets  
25 or shelves designed to promote convective circulation of air between the compartments also promote the circulation of moisture, enzymes and harmful bacteria. In addition, any liquid that may spill or leak, such as juices running from uncooked meats, will not be contained by the open baskets or shelves.

30 Conventional cold-storage appliances exemplified by upright refrigerators and chest freezers are not the only prior art disclosures of interest. For example, it has been known for many years to divide a refrigerator into compartments, each with its own dedicated

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door or lid. Examples of this idea are disclosed in UK Patent Nos. GB 602,590, GB 581,121 and GB 579,071, all to Earle, that describe cabinet-like refrigerators.

5 In those Earle documents, the front of the cabinet is provided with a plurality of rectangular openings for receiving drawers. Each drawer has a front panel larger than its respective opening so that a vertical seal is formed around the overlap when the drawer is in a closed position. The drawers and their contents are cooled by a cooler unit that circulates cooled air by convection within the cabinet, in common with the types of refrigerator already described. To promote circulation of this air amongst all of the  
10 drawers, the drawers are open-topped and have apertures in their bottoms. Also, the drawers are disposed in a stepped arrangement, those at the top of the refrigerator extending back less far into the cabinet than the lower drawers so that the rear of each drawer is exposed to the downward flow of cooled air from the cooler unit.

15 Although only one drawer need be opened at a time, the apertures in the bottom allow cold air to flow freely from the open drawer, which is replaced by warm moist ambient air to the detriment of energy efficiency and with the increased possibility of cross-contamination. Indeed, when a drawer is opened, cold air within the cabinet above the level of that drawer will flood out, drawing ambient air into the cabinet. Furthermore, the  
20 drawers encourage ambient air to flow into the interior of the refrigerator because, upon opening, they act as pistons drawing the ambient air into the interior of the refrigerator cabinet. Once in the cabinet, the warm air can circulate as freely as the cold air that is supposed to be there.

25 Even when closed, the accumulation of cold air towards the bottom of the cabinet will exert increased pressure on the vertical seals of the lowest drawers, increasing the likelihood of leakage if the seal is faulty.

A further example of the above type of refrigerator is disclosed in UK Patent No.  
30 GB 602,329, also to Earle. The refrigerator disclosed therein suffers many of the above problems but is of greater interest in that a single drawer consisting of insulated sides and base is provided within the cooled interior of the cabinet. In contrast to the variants

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outlined above, the sides and base are solid and not perforated so that air cannot flow through them. When the drawer is closed, a horizontal member within the cabinet combines with the drawer to define a compartment, the horizontal member thus being a lid for the drawer. This compartment is provided with its own cooling coils situated just  
5 below the horizontal member.

Very little detail is given about the seal that is formed between the drawer and the horizontal member, other than that the horizontal member has a downwardly projecting rear end with a biased edge that makes a close fit with the rear wall of the drawer. Nothing  
10 else is said about the junction between the drawer and the horizontal member, apart from the general statement that the drawer is adapted when in its closed position to fit 'fairly snugly' against the horizontal member. It can only be inferred that the drawer and the horizontal member merely abut against each other. Whilst this will impede the passage of air into and out of the drawer, it will not form an impervious seal. As this is not a vapour  
15 seal, icing and cross-contamination is likely to occur even when the drawer is closed.

The drawer arrangement described creates a compartment in which a different temperature can be set when compared to the essentially common temperature of the rest of the refrigerator. It is particularly envisaged that the drawer can act as a freezer compartment.  
20 The Applicant has appreciated a disadvantage in this arrangement, namely that as the freezer drawer resides within the cooled interior when closed, the outer surfaces of the drawer within the cabinet will be cooled to the temperature of the refrigerator. Accordingly, when the drawer is opened, those cooled outer surfaces will be exposed to ambient air containing moisture that will condense on the cooled surfaces leading to an  
25 undesirable accumulation of moisture. Condensation involves transfer of latent heat from water vapour to the drawer, thus increasing the burden of cooling the drawer again when the drawer is returned to the closed position within the cabinet.

Additionally, condensed moisture will be transferred to the interior of the refrigerator  
30 when the drawer is closed. As discussed above, the presence of water promotes microbial activity. A further disadvantage of introducing water into the interior of the refrigerator is that it may freeze: this can be a particular problem where the drawer of the enclosed

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compartment meets the insulated top, as any ice formation will form a seal that locks the drawer in a permanently closed position. In fact, the of ice formation is due to moisture migration across the interface between the drawer and the top. This disadvantage was appreciated by Earle, as a cam mechanism is mentioned in GB 602,329 to break any ice  
5 formed at the seals or on the runners or other support surfaces of the drawers. It is also possible for a build-up of ice to affect the sealing ability of the seal, by preventing mating sealing surfaces from mating correctly. Of course, the accumulation of ice on moving parts of the drawer mechanism is also undesirable as it will impede movement of the drawer.

10 A further interesting prior art document, cited as technological background against WO 01/20237, is US Patent No. 1,337,696 to Ewen. Ewen speaks of segregation between refrigerated drawers contained in a surrounding cabinet and employs refrigerating units placed 'immediately and closely above each drawer... so that said drawer may in effect be said to be closed against said refrigerating unit'. However, there has to be a gap left  
15 between the drawer and the refrigerating unit if the drawer is going to open. As in Earle, that gap will promote icing as moist air within the cabinet migrates into the drawer and the water vapour condenses and freezes. The smaller the gap, the sooner the accumulating ice will prevent drawer movement. If a larger gap is tried instead, there will be a greater spillage of air and hence the refrigerator will be less energy-efficient and more susceptible  
20 to cross-contamination.

That aside, the spillage of cold air in Ewen lowers the temperature within the cabinet around the drawers, and so increases the likelihood of condensation on the drawers when opened. It will be noted that cold air spilled in this way can fall freely behind the drawers  
25 within the cabinet and so expose the exterior of the drawers to air substantially below ambient temperature. Certain design details of Ewen worsen this effect. For example, the bottom wall of the Ewen unit is an efficient insulator which will significantly reduce the surface temperature of the drawers. Also, the internal divisions between the drawers do not allow for ambient heat transfer to the drawers but only for heat transfer between the  
30 drawers, thus promoting drawer-to-drawer temperature equalisation over time. Left for long periods, or even overnight, large parts of the external surface of each drawer will fall to temperatures significantly below ambient dew point. Condensation or ice will therefore



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form on those surfaces as soon as the drawers are opened; similarly, if the drawers are removed and left outside the appliance, they will start to ‘sweat’ with condensation.

Like Earle, opening and closing a drawer within a sealed cabinet in Ewen acts like a piston, alternately applying both negative and positive pressures to adjacent areas. This promotes air transfer through the drawer opening at the front of the cabinet, which can displace cold treated air in a drawer, and within the cabinet itself. An over-sized cabinet would reduce the piston effect but would also be wasteful of space. Conversely, a more space-efficient close-fitting cabinet may decrease the displacement of cold treated air, and so reduce the burden of cooling the warmer air that takes its place, but it will increase resistance to opening and closing the drawer.

Cold air spillage aside, the gap inevitably left between a drawer and its associated lid in prior art arrangements is large enough to allow the passage of enzymes, spores and other airborne contaminants. Also, Ewen discloses a common interconnecting drain and this too would allow free transfer of contaminants between each drawer, particularly under the aforementioned piston action.

Whilst Ewen speaks of different temperatures in different drawers, the plurality of cooling lids are connected in series and have no means for individual temperature control in each drawer. The different temperatures are designed-in by providing some drawers with more cooling elements than others, but there is no measurement or control of those temperatures in use. Also, like the compartments of more conventional prior art, each drawer in Ewen has a fixed function, namely freezer or refrigerator.

Even if removed from the appliance, Ewen’s drawers will stay attached to their drawer fronts and runners. This does not lend the drawers to temporary storage or transport. Moreover, like Earle, the drawers in Ewen cannot be opened fully: they can only be opened less than half-way while being supported by the structure of the appliance. This is to the detriment of access to, and visibility and illumination of, the contents.

Finally, a more recent prior art document, DE 19546984, discloses a cold-storage

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appliance in which various means are provided to prevent loss of cold air when drawers are opened and/or removed. These means include seals around the drawer opening combined with a hinged flap that drops down to close the drawer opening when a drawer is removed, or a plate or similar member at the back of a drawer that moves with the drawer and blocks the opening when the drawer is removed. However, none of these provisions reduce or prevent air flowing and intermixing within the cabinet of the appliance under the piston effect of a drawer opening and closing. For example, the drawers are all open-topped and held within a common cooled cabinet. Also, to the extent that airflow is restricted, the drawers will encounter great resistance to being opened and closed.

It is against this background that the present invention has been devised.

From one aspect, the invention resides in a cold-storage appliance including: at least one container; a structure defining a vapour-sealable container compartment from which the container can be withdrawn to open the container and afford access to its interior and to which the container can be returned to close the container for cold storage of any items within the container; and valve means communicating with the container compartment to admit air from outside the appliance when the container is being withdrawn from the container compartment, to expel air outside the appliance when the container is being returned to the container compartment, and to maintain a vapour seal with the container compartment when the container is within the container compartment. This mitigates the piston effect which could otherwise promote cross-contamination between containers in a multi-container arrangement.

The invention also resides in a cold-storage appliance including: at least one container; a structure defining a vapour-sealable container compartment from which the container can be withdrawn to open the container and afford access to its interior and to which the container can be returned to close the container for cold storage of any items within the container; and barrier means adapted to advance and to retract along the container compartment as the container is being withdrawn from and returned to the container compartment, while maintaining a vapour seal so as (i) to separate from the interior of the

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container air drawn into the container compartment by withdrawal of the container and (ii) to displace that air from the container compartment when the container is returned to the container compartment. This also mitigates the piston effect and its risk of cross-contamination by maintaining a vapour seal between the interior of the container and air  
5 within the compartment vacated by the container. For example, the barrier means can be a bellows extensible along and sealed to the surrounding walls of the container compartment.

In order that the present invention can be more readily understood, reference will now be  
10 made, by way of example only, to the accompanying drawings in which:

Figure 1 is a front view of a refrigerator/freezer appliance as disclosed in the Applicant's co-pending International Patent Application No. PCT/GB00/03521 (WO 01/20237), showing a vertical array of drawers each including a bin;  
15

Figure 2 is a side view of the appliance of Figure 1, with a lower portion of a side panel removed so that the sides of the drawers can be seen;

Figure 3 is a section along line III-III of Figure 2 but with the drawers closed;  
20

Figure 4 is a section along line IV-IV of Figure 1;

Figures 5(a) and 5(b) are a plan view and a sectional side view respectively of a lid showing its sealing, cooling and drainage facilities in detail;  
25

Figure 6 is a diagrammatic view of a plurality of the lids of Figures 5(a) and 5(b), showing their separate drainage arrangements;

Figures 7(a) and 7(b) are a bottom plan view and a sectional side view respectively of a lid adapted for use in a fan coil cooling system;  
30

Figures 8(a), 8(b) and 8(c) are front and side elevation views and an enlarged

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partial cross-sectional detail view of a bench-type cold-storage appliance having an alternative layout to that shown in Figures 1 to 4;

5       Figures 9(a), 9(b) and 9(c) are a front elevation and two sectional views respectively of a cold-storage appliance that is functionally akin to the aforementioned Ewen prior art but addresses many of its problems;

10       Figures 10(a), 10(b), and 10(c) are partial sectional side views showing various ways of preventing temperature gradients within each compartment of the appliance of Figures 9(a), 9(b) and 9(c);

15       Figures 11(a), 11(b), and 11(c) are schematic sectional side views of a hinged-plate air transfer valve for use in the appliance of Figures 9(a), 9(b) and 9(c), in three modes of operation;

      Figures 12(a), 12(b), and 12(c) correspond to Figures 11(a), 11(b), and 11(c) but show a loose-plate air transfer valve in the same three modes of operation;

20       Figures 13(a), 13(b), and 13(c) correspond to Figures 11(a), 11(b), and 11(c) and Figures 12(a), 12(b), and 12(c) but show a diaphragm air transfer valve in the same three modes of operation; and

25       Figure 14 is a sectional side view of two drawers of the appliance of Figures 9(a), 9(b) and 9(c) fitted with bellows to minimise air transfer into the drawer compartments during drawer opening and closing.

30       Whilst the disclosure of the Applicant's co-pending International Patent Application No. PCT/GB00/03521 (WO 01/20237) is incorporated herein by reference, Figures 1 to 4 of WO 01/20237 are reproduced in the drawings appended to this specification and will now be described to help put the present invention into context.

Figures 1 to 4 show a refrigerator/freezer appliance 2 according to WO 01/20237. The

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appliance 2 is of upright cuboidal configuration, and comprises five rectangular-fronted drawers 4 arranged one above another and housed in a cabinet 6 comprising top 8, bottom 10, side 12 and rear 14 panels. Any of these panels can be omitted if it is desired to build the appliance 2 into a gap between other supporting structures; in particular, the side  
5 panels 12 can be omitted if neighbouring cupboards can be relied upon for support or otherwise to perform the function of the side panels 12. The panels 8, 10, 12, 14 may or may not be structural but if they are not, a frame (not shown) provides support for the various parts of the appliance. If a frame is provided, it is structurally unnecessary to have panels.

10

The drawers 4 can be slid horizontally into and out of the cabinet 6 by means of tracks or runners on the sides of the drawers 4 that will be described in more detail below. If there is no back panel 14, it is theoretically possible for a drawer 4 to be removed from the cabinet 6 in more than one direction, as shown in Figure 2.

15

Each drawer 4 comprises an insulated open-topped bucket-like container 16, at least one container 16 (in this case, that of the central drawer 4) being of a different depth to the other containers 16 to define a different internal volume. These containers 16 will be referred to in this specific description as storage bins or more simply as bins 16. The  
20 bottom bin 16 leaves only a narrow gap to the bottom panel 10 of the cabinet 6, whereas the top bin 16 leaves a substantial space at the top of the appliance 2 under the top panel 8, allowing room for a compartment 18 that accommodates a refrigerator engine 20, for example including condenser and compressor means as is well known.

25 The relatively deep bin 16 of central drawer 4 is intended to hold bottles and other relatively tall items stored upright, whereas the other, relatively shallow bins 16 are for correspondingly shallower items. Compared to the shelves and other compartments defining the main storage volume of a conventional upright cold-storage appliance, all of the bins 16 have a favourable aspect ratio in terms of the substantial width of the access  
30 opening compared to the depth of the compartment thereby accessed. It is therefore very easy to reach every part of the interior of a bin 16 when a drawer 4 is opened,

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The interior of the cabinet 6 is divided by five insulated lids 22, one for each drawer 4, that are generally planar and horizontally disposed. When a drawer 4 is closed, the open top of its associated bin 16 is closed by an appropriate one of the lids 22 in a manner to be described. The lids 22 include cooling means 24 being evaporator elements of known type  
5 disposed in the lower face 26 of each lid 22 to cool the contents of a bin 16 closed by that lid 22.

Each bin 16 has a generally flat front face 28 that is exposed when the drawer 4 is closed. The front face 28 could be provided with a decorative panel as is well known. When the  
10 drawer 4 is closed, the front face 28 of the bin 16 is bordered at the top by a control and display panel 30 dedicated to that bin 16, the panel 30 being co-planar with the front face 28. The panel 30 is supported by the front edge 32 of the appropriate lid 22, the panel 30 being recessed into the front edge 32 of the lid 22.

15 The control and display panel 30 contains a number of displays, switches and audible alarms, thus providing a user interface for each bin 16. For example, the interface will most commonly be used for selecting the temperature to which the bin 16 is to be cooled, but also contains temperature displays, on/off and fast-freeze switches, a light indicating when the drawer 4 is open and an audible alarm to indicate when the drawer 4 has been  
20 open longer than a predetermined time or when the temperature inside the bin 16 has reached an upper or lower threshold.

A rounded handle 34 extends across substantially the entire width of the top portion of the front face 28 to enable the drawer 4 to be pulled out when access to the interior of the bin  
25 16 is required.

The bottom of the front face 28 of each bin 16 is bordered by a slot 36 that, as will be described, admits ambient air into the cabinet 6. To do so, each slot 36 communicates with an air gap 38 extending beneath the entire bottom face 40 of the associated bin 16 to meet  
30 a void 42 maintained behind each bin 16, the void 42 being defined by the inner surfaces of the back 14 and side 12 panels of the cabinet 6 and the backs 44 of the bins 16. As can be seen particularly from Figure 4, the void 42 extends behind each bin 16 from the base

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panel 10 of the cabinet 6 to communicate with the refrigerator engine compartment 18 at the top of the cabinet 6.

5 The air gaps 38 beneath the bins 16 and the void 42 behind the bins 16 also communicate with air gaps 38 to the sides 48 of the bins 16. Optionally, vents 46 are provided in the side panels 12 of the cabinet 6 adjacent to the bins 16 through which ambient air can also be admitted. As best illustrated in Figures 3 and 4, air gaps 38 extend around all bar the top side of each bin 16, so that ambient air entering the cabinet 6 through the slots 36 can circulate freely around the sides 48, bottom 40 and rear 44 of each bin 16. It will also be  
10 noted that ambient air can circulate freely over the top surface 50 of each lid 22. To allow this airflow over the uppermost lid 22, which does not have a bin 16 above, a slot 36 is provided under the front face 52 of the refrigerator engine compartment 18.

It will be noted that the piston action created by opening a drawer 4 that sucks ambient air  
15 into the interior of the appliance 2 does not pose a problem in this invention. In fact, this action is advantageous as it promotes circulation of ambient air within the cabinet 6.

Figure 4 shows that the refrigerator engine compartment 18 includes an impeller 54 exhausting through apertures 56 provided in the front face 52 of the refrigerator engine  
20 compartment 18. As best seen in Figure 1, these apertures 56 extend horizontally across the width of the front face 52. The impeller 54 communicates with the void 42 behind the bins 16 to draw air from the void 42, thus continuously promoting the induction of ambient air through the slots 36 and the optional side vents 46. Upon entering the refrigerator engine compartment 18, this air is drawn through the heat-exchange matrix 58  
25 of the condenser.

Accordingly, ambient air entering the cabinet 6 through the front slots 36 and, if provided, the side vents 46, leaves the cabinet 6 through the apertures 56 provided in the front face 52 of the refrigerator engine compartment 18, and so ambient air is circulated through the  
30 cabinet 6. More specifically, ambient air enters the appliance 2 where it immediately comes into contact with the outer surfaces 40, 44, 48 of the bins 16 and warms them to ambient temperature (or substantially so, as a surface resistance effect means that a sub-

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ambient boundary layer will remain due to the temperature gradient across the thickness of the bin wall) before being drawn towards the void 42 and then upwards through the void 42 by the circulation of the air. The arrows of Figure 4 demonstrate this circulation of air through the appliance 2. Accordingly, the interior of the cabinet 6 is kept close to ambient  
5 temperature, and only the interior of each bin 16 is cooled.

By exposing the external surfaces 28, 40, 44, 48 of the bin 16 to warmer air than it contains, there is no problem with condensation on the external surfaces 28, 40, 44, 48, and hence no problem with latent heat transfer to the bin 16 or the icing and cross-  
10 contamination difficulties of condensed water entering the cabinet 6.

In any event, cross-contamination would be unlikely to occur because each bin 16 is tightly sealed when its drawer 4 is closed. So, even if microbes enter the cabinet 6, they cannot readily gain access to other bins 16. It is also unlikely that two bins 16 would be  
15 open together at any given time. It would be possible to include means for enforcing this, for example using a mechanism akin to that used in filing cabinets for anti-tilt purposes, by preventing more than one drawer 4 being opened at a time. Such a mechanism will be described later.

20 When a bin 16 is open, its open top does not suffer much spillage of cold air, and when a bin 16 is closed, the horizontal seals 60 apt to be used in the invention are inherently better at sealing-in cold air than the vertical seals commonly used in upright refrigerators and freezers. Whilst horizontal seals are known in chest freezers, this invention does not suffer the inconvenience and space problems of chest freezers, instead being akin in those  
25 respects to the much more popular upright appliances. The seals 60 can have magnetic qualities, for example being operable by permanent magnets or electromagnets, or may employ hydraulics or pneumatics to expand or contract them.

As there has to be a large temperature gradient between the cooled inner surfaces 62 of  
30 each bin 16 and its outer surfaces 28, 40, 44, 48, the bins 16 are constructed from an efficient insulating material so that the gradient is easily maintained with the outer surfaces 28, 40, 44, 48 remaining close to the ambient temperature. Materials such as phenolic



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foam or polyurethane foam (optionally skinned with GRP or a polycarbonate in a composite structure) are particularly preferred for the construction of the bins 16.

5 If segregation of the contents of a particular bin 16 is required, that bin 16 may be fitted with removable inserts 64. The inserts 64 are of varying shape and dimensions and may be used to define many types of compartments. For instance, an insert 64 may be a thin partition with a length corresponding to the length or width of the bin 16 in which it is received. An insert 64 may be a box, with or without a lid, or an insert 64 may include clips for holding bottles in place or trays for holding eggs or the like. An insert 64 could  
10 also be a wire basket or shelf.

As can be seen in Figure 2, one or more of the bins 16 can be removed from the appliance 2 and fitted with an insulated transport cover 66. The bin 16 may then be taken away from the appliance 2, its insulated construction ensuring that it keeps its contents cool for a  
15 limited period of time. For instance, the bin 16 may be used as a cool-box, possibly in conjunction with ice-packs to keep the interior cool for as long as possible. Alternatively, the bin 16 with transport cover 66 may be kept close to the appliance 2 to provide added temporary cooled storage capacity, further bins 16 being fitted to the appliance 2 in that event. Further details of transport cover arrangements will be given later.

20

It is also possible for a transport cover 66 to include a refrigerator engine powered internally by batteries or a gas supply or externally by mains electricity or a vehicle electricity supply.

25 Although not shown in the general views of Figures 1 to 4, the Applicant's co-pending International Patent Application No. PCT/GB00/03521 (WO 01/20237) discloses ways in which a bin 16 can be moved with a major horizontal component of movement to gain access to the interior of the bin 16 and, during that access movement, also with a minor vertical component of movement to clear the lid 22.

30

Figures 5(a) and 5(b) show preferred details of the lids 22 to which the bins 16 seal when fitted in the appliance 2. Figure 5(a) shows that the lid 22 is oblong in plan view. The

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oblong dashed outlines of features below the lid 22 are also apparent. Starting inwardly and moving outwardly, those features are an evaporator 194 disposed centrally on the underside of the lid 22, a drain pan 196 disposed beneath the evaporator 194 to catch water that drips from the evaporator 194, and a recess 198 in the underside of the lid 22 that  
5 accommodates both the drain pan 196 and the evaporator 194.

As best appreciated from Figure 5(b), which is a cross-section on line A-A of Figure 5(a), the recess 198 is bounded by a peripheral skirt 200 depending from the lid 22. A pair of oblong compressible seals 60 lie one within the other on the lower end face 202 of the  
10 skirt 200. Those seals 60 are continuous save for an opening that accommodates an oblong-section drainage duct 204 leading rearwardly from the drain pan 196. The drain pan 196 has an inclined base 206 to lead water toward that drainage duct 204, from which the water is channelled away from the lid 22 as Figure 6 will explain. A temperature sensor (not shown) can penetrate the skirt 200 above the seals 60 to measure the  
15 temperature within the cavity sealed by the bin 16 and the lid 22.

Figure 6 shows how it is preferred that separate drain ducts 208 run from each drain pan 196 of a multi-bin appliance 2. This minimises the risk of cross-contamination. Each duct 208 includes a U-bend 210 defining a sealing water trap and drains separately to a  
20 common tray 212. That tray 212 may be located above a compressor 214 of the appliance 2 as shown so that, over time, heat emanating from the compressor 214 evaporates the water from the tray 212 at least as quickly as that water accumulates in the tray 212. In addition or in the alternative, the condenser fan of the appliance 2 (not shown) can blow across the surface of water in the tray 212 so as to promote its evaporation.

25

Figures 7(a) and 7(b) show a further lid design suitable for use in a fan coil cooling system in which air is supplied to the bin 16 and extracted from the bin 16 via a remote fan coil unit. Such a system is also known as a forced air system, and the lid 22 in Figures 7(a) and 7(b) is hollow and partitioned to govern the flow of air on which such systems depend.  
30 Thus, cold air cooled by a heat exchanger (not shown) is piped under pressure from a fan (not shown) into a supply air plenum 216 disposed peripherally within the lid 22, from which that air enters the bin through supply air diffusion slots 218 around a base panel 220

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defining the underside of the lid 22. Warmer air is extracted from the bin 16 through a centrally-disposed return air plenum 222 that communicates with the bin 16 through a central hole 224 in the base panel 220 and with the fan through a pipe 226 extending through the surrounding supply air plenum 216. The warmer air is drawn into the return air  
5 plenum 222 under low pressure created by the fan, and is then sent to the heat exchanger to be cooled and recirculated via the supply air plenum 216.

Apart from the vertical array of drawers 4 common to the embodiments described above, a side-by-side arrangement of drawers 4 is also contemplated as shown in Figures  
10 8(a), 8(b) and 8(c). The front view of Figure 8(a) shows a four-drawer bench-type appliance 268 in which the drawers 4 are in two adjacent columns of two drawers 4 each. Thus, the appliance 268 is low enough to have a worktop 270 over the drawers 4, straddling the two columns. This embodiment of the invention is therefore suitable for use as a refrigerated food preparation and/or servery unit.

15

The depth of the drawers 4 is maximised within the limited available height by mounting the refrigerator engine 272 and control panel 274 in a side-slung position to one side of the appliance 268 as shown. Also, the side view of Figure 8(b) and the enlarged detail cross-sectional view of Figure 8(c), taken on line X-X of Figure 8(a), shows that the front edge  
20 of the worktop 270 has a raised lip 276 that helps to prevent spillages on the worktop 270 dripping down onto or into the drawers 4 below.

Figures 8(a) and 8(b) also show how an appliance 268 of the invention can be mounted on castors 278; those castors 278 can be height-adjustable to level the appliance 268 on a  
25 non-level floor 280.

Referring now to Figure 9(a), this is a front elevation of a cold-storage appliance 332 that is functionally akin to the aforementioned Ewen prior art - in that much of the bin exterior may be exposed to air below ambient temperature - but addresses many of its problems. In  
30 outward appearance, the appliance of Figure 9(a) is similar to the bench-type appliance shown in Figures 8(a) and 8(b) and has its drawers 4, control panel 274 and refrigerator engine 272 in similar positions under a worktop 270. Similarly, Figure 9(b) - a section on

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line A-A of Figure 9(a) - shows that each drawer 4 comprises a lid 22 within the cabinet and a bin 16 movable forwardly from the cabinet on telescopic runners 74. Figure 9(c) - a section on line B-B of Figure 9(a) - shows those runners 74 beside the bin 16.

5 However, like Ewen and unlike the preceding embodiments, there is no provision for vertical movement of the bin 16 with respect to the lid 22: instead, the bin 16 simply slides horizontally past the close-fitting lid 22. The gap between the bin 16 and the lid 22 that is therefore necessary requires the creation of a complete external vapour barrier around the compartment that houses each drawer, so as to overcome the associated moisture  
10 transmission and cross-contamination problems. Thus, as Figure 9(b) shows, the front panel 118 of each drawer extends beyond the corresponding drawer aperture defined between the lids 22 or between a lid 22 and a bottom panel 334. The overlapping portion of each front panel 118 has vertical magnetic seals 336 on its rear face that, when the drawer 4 is closed, seal by flexible resilience and magnetic attraction against the opposed  
15 front faces of the lids 22 and/or the bottom panel 334 as appropriate. The drawer compartments are also sealed from each other; this applies especially to penetrations and service connections that run between neighbouring drawer compartments. It also applies to water drainage from the drawer compartments: separate drain lines (not shown) should be run individually from each compartment and should each have in-line water traps, akin to  
20 the drainage arrangement shown in Figure 6 above.

Figure 9(c) also shows L-section bin supports 338 depending from the runners 74 to cradle the bin 16 in a removable manner. Each drawer compartment should be of the minimum volume that allows access to the bin 16 and the associated runners 74.

25 Ideally, the walls of each drawer compartment should be of a thin, poorly-insulating material with all of the external surfaces exposed to ambient air. That ideal is not practically achievable, although use of a thin material that conducts heat from outside the drawer compartment and promotes convection of heat within the drawer compartment is  
30 to be encouraged. In reality, a temperature gradient will form over time inside the sealed compartments due to the insulating effect of the external enclosure and the air cavity around the bin 16, coupled with the refrigerating effect of the gap between the bin 16 and

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the lid 22. Additionally, an insulation trap will occur where external surfaces of a compartment are adjacent to other compartments. This further cools the air around the bin 16 and so increases the temperature gradient.

- 5 For this reason, Figures 10(a), 10(b), and 10(c) propose three different ways of minimising the effects of temperature gradients and cooling within each drawer compartment of the appliance of Figures 9(a), 9(b) and 9(c). Figure 10(a) shows a circulating fan 340 in a duct 342 behind the drawer compartment 344 that draws air from and pumps air back into the compartment 344. The circulation thus created in the air within the compartment 344
- 10 keeps the external surfaces of the bin 16 at an even temperature. Figure 10(b) shows an electric or hot gas heater 346 below the bin 16 to create convection currents in the air within the compartment 344 and to warm most of the external surfaces of the bin 16 to near-ambient, ambient or even higher than ambient temperature. Figure 10(c) shows air gaps 348 akin to those described above with reference to Figures 1 to 4, save that the air
- 15 gaps 348 are around and between the drawer compartments 334 rather than around the bin 16 itself. Although not visible in Figure 10(c), the air gaps 348 can extend over the sides of the drawer compartment 334 as well as the underside and back of that compartment 334 as shown. Again, air within the air gaps 348 can be warmed if needs be.
- 20 The next group of drawings - Figures 11 to 13 - show various air transfer valves that mitigate the piston effect experienced when opening and closing a drawer 4 of the appliance 332 of Figures 9(a), 9(b) and 9(c). Figures 11(a), 12(a) and 13(a) show the status of the respective valves when the bin 16 is static, for example when the bin 16 is closed. Figures 11(b), 12(b) and 13(b) show the status of the respective valves when the bin 16 is
- 25 being opened, and Figures 11(c), 12(c) and 13(c) show the status of the respective valves when the bin 16 is being closed.

Dealing firstly with the features in common to all of the valves of Figures 11 to 13, they comprise an inlet duct 350 closable by an inlet valve member 352, an exhaust duct 354

30 closable by an exhaust valve member 356, and an intermediate duct 358 connected to the drawer compartment 344 that communicates with a central valve chamber 360 disposed between the valve members 352, 356. The inlet valve member 352 normally rests on an

– 20 –

inlet valve seat 362 around a port 364 between the inlet duct 350 and the central valve chamber 360, and opens into the central chamber 360. Similarly, the exhaust valve member 356 normally rests on an exhaust valve seat 366 around a port 368 between the exhaust duct 354 and the central valve chamber 360, but the exhaust valve member 356  
5 opens into the exhaust duct 354.

Both of the valve members 352, 356 are in their respective rest positions when the bin 16 is static, as Figures 11(a), 12(a) and 13(a) show. When the drawer 4 is being opened, however, air is drawn from the central valve chamber 360 toward the drawer compartment  
10 344 as shown in Figures 11(b), 12(b) and 13(b). The resulting low pressure in the central valve chamber 360 causes the inlet valve member 352 to open and admit air from the inlet duct 350 through the inlet port 364 into the central valve chamber 360 and from there into the drawer compartment 344. Conversely, when the drawer 4 is being closed, air is forced from the drawer compartment into the central valve chamber 360 as shown in Figures  
15 11(c), 12(c) and 13(c), which increases the pressure in the central valve chamber 360 and so causes the exhaust valve member 356 to open. The air in the central valve chamber 360 thereby exhausts through the exhaust port 368 and out of the exhaust duct 354.

The inlet duct 350 draws ambient air from outside the cabinet of the appliance 332 and the  
20 exhaust duct 354 similarly exhausts air to outside the cabinet. Thus, such air as is displaced by the piston action of the opening and closing drawer 4 does not travel between compartments 344 of the appliance 332 and so does not have a chance to cross-contaminate between those compartments 344. It will be noted that in the passive or rest position shown in Figures 11(a), 12(a) and 13(a), the valves maintain the vapour barrier of  
25 the cabinet. The valves and the inlet and exhaust ducts 350, 354 should be sized and located to minimise disturbance to the cold treated air in the bin 16 as the bin 16 experiences piston action upon opening and closing the associated drawer 4.

The differences between the valves of Figures 11 to 13 lie mainly in the valve members  
30 and in their sealing arrangements with respect to their seats. The valve members 352, 356 in Figures 11 and 12 are rigid flaps or plates having magnetic or compressible seals 370 around their periphery that co-operate with the valve seats 362, 366 to seal the respective

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ports 364, 368. Both variants rely upon gravity and pressure differences to close the respective ports 364, 368. In Figure 11, the plates 352, 356 are hinged to one side so as to lift to the other side when they open their ports 364, 368. In Figure 12, however, the plates 352, 356 lie loosely on their seats 362, 366 and may be inclined as shown to encourage one side to lift in favour of the other side when they open their ports 364, 368. In contrast, the valve members 352, 356 of Figure 13 are curved diaphragms whose curvature and resilience biases them shut, but which flex resiliently in response to pressure changes to open by flattening against the bias. It will also be evident from Figure 13 that the diaphragms 352, 356 have a flexible peripheral flange 372 that seals against the valve seats 362, 366. The diaphragm valve of Figure 13 has the benefit that its operation does not rely upon gravity to any extent; hence, the valve can be oriented in any way that may be desired.

Another way to minimise cross-contamination between drawer compartments 344 is to maintain a vapour barrier effective throughout the range of movement of a drawer 4. Figure 14 shows a way of achieving this and depicts two drawers 4 of an appliance, one above the other. The upper drawer 4 is closed and the lower drawer 4 is open and it will be noted that each is fitted with a bellows 374 attached to the rear face of the bin 16 in each drawer 4. The cross-section of each bellows 374 substantially matches the oblong cross-sectional shape and size of the drawer compartment 344, and each is capable of extending the full length of the drawer compartment 344 swept by the bin 16. Thus, using a piston and cylinder analogy, the bellows 374 sweeps substantially all of the volume of the associated drawer compartment 344.

The bellows 374 attached to the bin 16 of the upper drawer 4 is collapsed compactly behind the bin 16 in the closed upper drawer 4 but the corresponding bellows 374 of the open lower drawer 4 is fully extended. It will be appreciated that each bellows 374 is closed at its bin end and open to its rear end, and that the rear end is sealed around its periphery to the surrounding walls of the drawer compartment 344 to form a vapour barrier. The open end of each bellows 374 communicates directly with the surrounding atmosphere so that fresh air is drawn into the bellows 374 as the drawer 4 opens and exhausts as the drawer 4 closes. However, there is no mixing between the incoming air in

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the bellows 374 and the air that is or was in the drawer compartment 344.

Whilst the solutions described in Figures 10 to 14 above are primarily intended to solve problems with refrigerators akin to the prior art exemplified by Ewen, some or all of those solutions can be applied to refrigerators designed in accordance with the Applicant's co-  
5 pending International Patent Application No. PCT/GB00/03521 (WO 01/20237).

Similarly, trace heating may be applied locally to the skirt depending from the lid so as to prevent icing at the bin/lid interface. The heating may be effected by electric elements or  
10 hot gas means, and may be continuous or applied only when it is desired to open the bin. Among many variations of the invention herein, the trace heating facility could be applied to any arrangement in which relative movement (preferably vertical movement) between the lid and the bin breaks the seal and clears the bin from the lid (or vice versa) so that the bin can be opened fully for access.

15

It is also possible for heat to be recovered from various heat-emitting parts of refrigerator engines, such as heat exchangers, motors and so on, and applied to warming the bin/lid interface or the external surfaces of the bin 16 within a drawer compartment. This saves energy by using waste heat that would otherwise be rejected to atmosphere.

20

The invention has wide-ranging applications and benefits for storing, handling, distributing, transporting and delivering items in the right condition, notably:

precise variable temperature and humidity control that could even include heating  
25 rather than merely cooling;

mechanical protection of stored items;

sterile storage with minimal risk of cross-contamination;

30

the option of storage in partial vacuum conditions;



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the option of storage in a preservative holding gas environment;

isolation of stored items against vibration and agitation; and

5           containment of, or protection against, radiation and bio hazards.

In general, therefore, reference should be made to the appended claims and other general statements herein rather than to the foregoing specific description as indicating the scope of the invention. In interpreting the invention, it should be understood that  
10   although features of the illustrated embodiments have been described in combination with each other and although such combinations may have advantages of their own, many of those features can be applied independently. So, those features are considered to be independently patentable whether within or beyond the inventive concepts expressed herein.

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**CLAIMS**

1. A cold-storage appliance including:

5           at least one container;

          a structure defining a vapour-sealable container compartment from which the container can be withdrawn to open the container and afford access to its interior and to which the container can be returned to close the container for cold storage of any items  
10       within the container; and

          valve means communicating with the container compartment to admit air from outside the appliance when the container is being withdrawn from the container compartment, to expel air outside the appliance when the container is being returned to the  
15       container compartment, and to maintain a vapour seal with the container compartment when the container is within the container compartment.

2. The appliance of Claim 1, wherein the valve means comprises an inlet duct closable by an inlet valve member, an exhaust duct closable by an exhaust valve member, and an  
20       intermediate duct connected to the container compartment.

3. The appliance of Claim 2, wherein the intermediate duct communicates with a central valve chamber disposed between the inlet valve member and the exhaust valve member.

25       4. The appliance of Claim 3, wherein the inlet valve member normally rests on an inlet valve seat around a port between the inlet duct and the central valve chamber, and opens into the central valve chamber.

30       5. The appliance of Claim 3 or Claim 4, wherein the exhaust valve member normally rests on an exhaust valve seat around a port between the exhaust duct and the central valve chamber, and opens into the exhaust duct.

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6. The appliance of any of Claims 2 to 5, wherein the inlet valve member and the exhaust valve member are in their respective rest positions when the container is static.
7. The appliance of any of Claims 2 to 6, wherein when the container is being opened, the  
5 inlet valve member opens and admits air from the inlet duct to flow into the container compartment.
8. The appliance of any of Claims 2 to 7, wherein when the container is being closed, air forced from the container compartment causes the exhaust valve member to open and is  
10 exhausted out of the exhaust duct.
9. The appliance of any of Claims 2 to 8, wherein the inlet duct draws ambient air from outside the appliance and the exhaust duct exhausts air to outside the appliance.
- 15 10. The appliance of any of Claims 2 to 9, wherein the valve members are rigid flaps or plates.
11. The appliance of any of Claims 2 to 10, wherein the valve members rely upon gravity and pressure differences to close.  
20
12. The appliance of any of Claims 2 to 9, wherein the valve members are resilient diaphragms whose resilience biases them shut.
13. The appliance of any of Claims 2 to 12, wherein the valve members have magnetic or  
25 compressible seals around their periphery that co-operate with valve seats to seal respective ports.
14. A cold-storage appliance including:
- 30 at least one container;
- a structure defining a vapour-sealable container compartment from which the

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container can be withdrawn to open the container and afford access to its interior and to which the container can be returned to close the container for cold storage of any items within the container; and

5            barrier means adapted to advance and to retract along the container compartment as the container is being withdrawn from and returned to the container compartment, while maintaining a vapour seal so as (i) to separate from the interior of the container air drawn into the container compartment by withdrawal of the container and (ii) to displace that air from the container compartment when the container is returned to the container  
10   compartment.

15. The appliance of Claim 14, wherein the barrier means is a bellows extensible along the surrounding walls of the container compartment.

15    16. The appliance of Claim 14 or Claim 15, wherein the barrier means is attached to the rear of the container.

17. The appliance of Claim 15 or Claim 16, wherein the bellows is closed at its forward  
20   end.

18. The appliance of any of Claims 15 to 17, wherein the bellows is open at its rearward  
end.

19. The appliance of Claim 18, wherein the rearward end of the bellows is sealed around  
25   its periphery to the surrounding walls of the container compartment to form a vapour barrier.

20. The appliance of Claim 18 or Claim 19, wherein the open end of the bellows  
communicates with the surrounding atmosphere so that atmospheric air is drawn into the  
30   bellows as the container opens and exhausts as the container closes.

21. The appliance of any of Claims 14 to 20, wherein the barrier means has a cross-section

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that substantially matches the cross-sectional shape and size of the container compartment.

22. The appliance of any of Claims 14 to 21, wherein the barrier means sweeps most or substantially all of the volume of the container compartment.

5

23. A cold-storage appliance including:

at least one container defining an external surface;

10 a structure defining a container compartment from which the container can be withdrawn to open the container and afford access to its interior and to which the container can be returned to close the container for cold storage of any items within the container; and

15 circulating means for circulating air around the container compartment to expose the external surface of the container therein to said circulating air.

24. The appliance of Claim 23, wherein the circulating means includes a fan.

20 25. The appliance of Claim 23 or Claim 24, wherein the circulating means includes a heater creating a convection current within the container compartment.

26. The appliance of Claim 25, wherein the heater is positioned below the container when the container is in the container compartment.

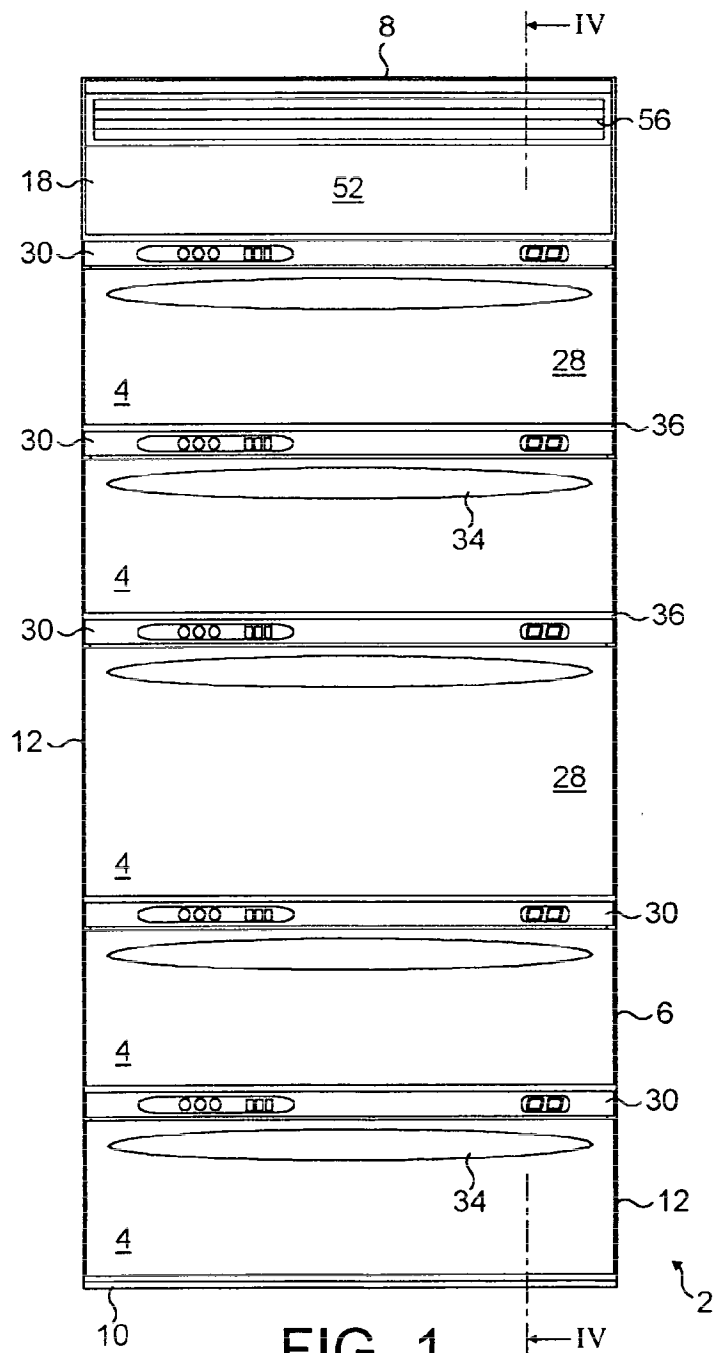


FIG. 1

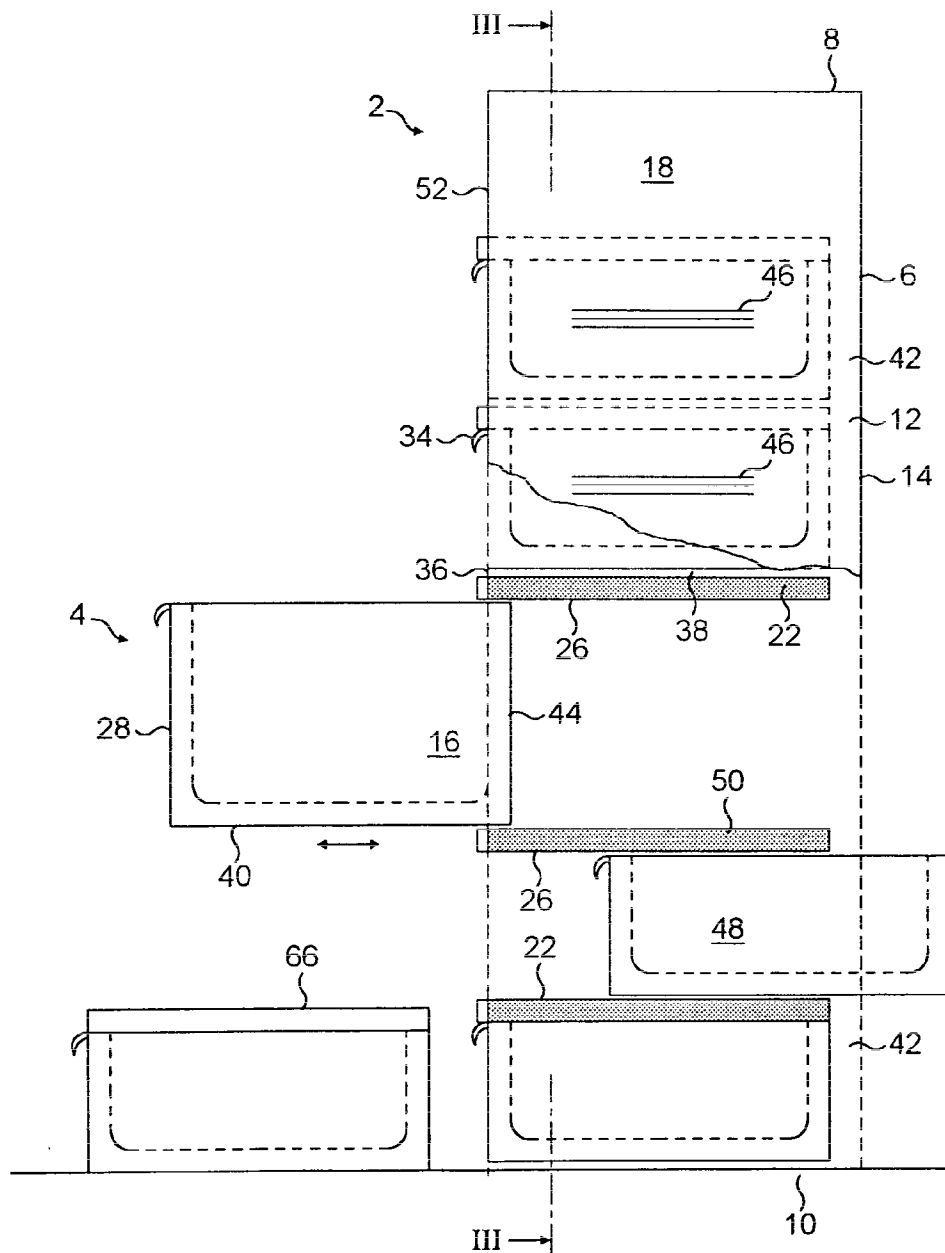


FIG. 2

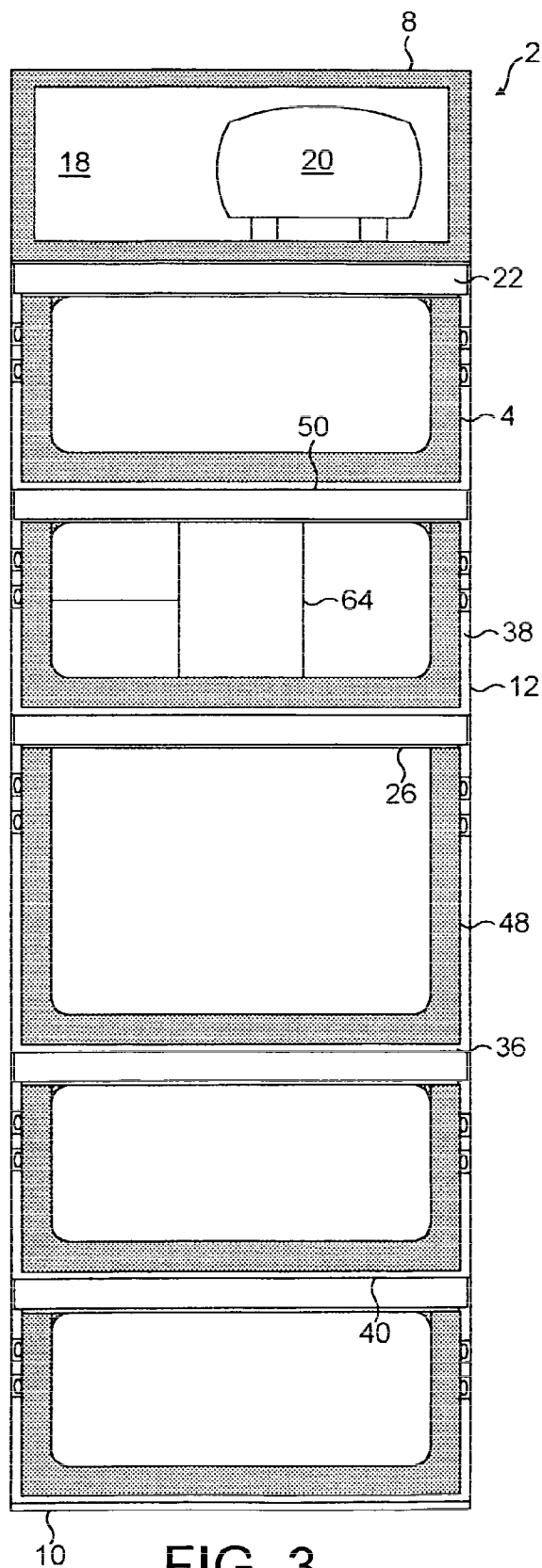
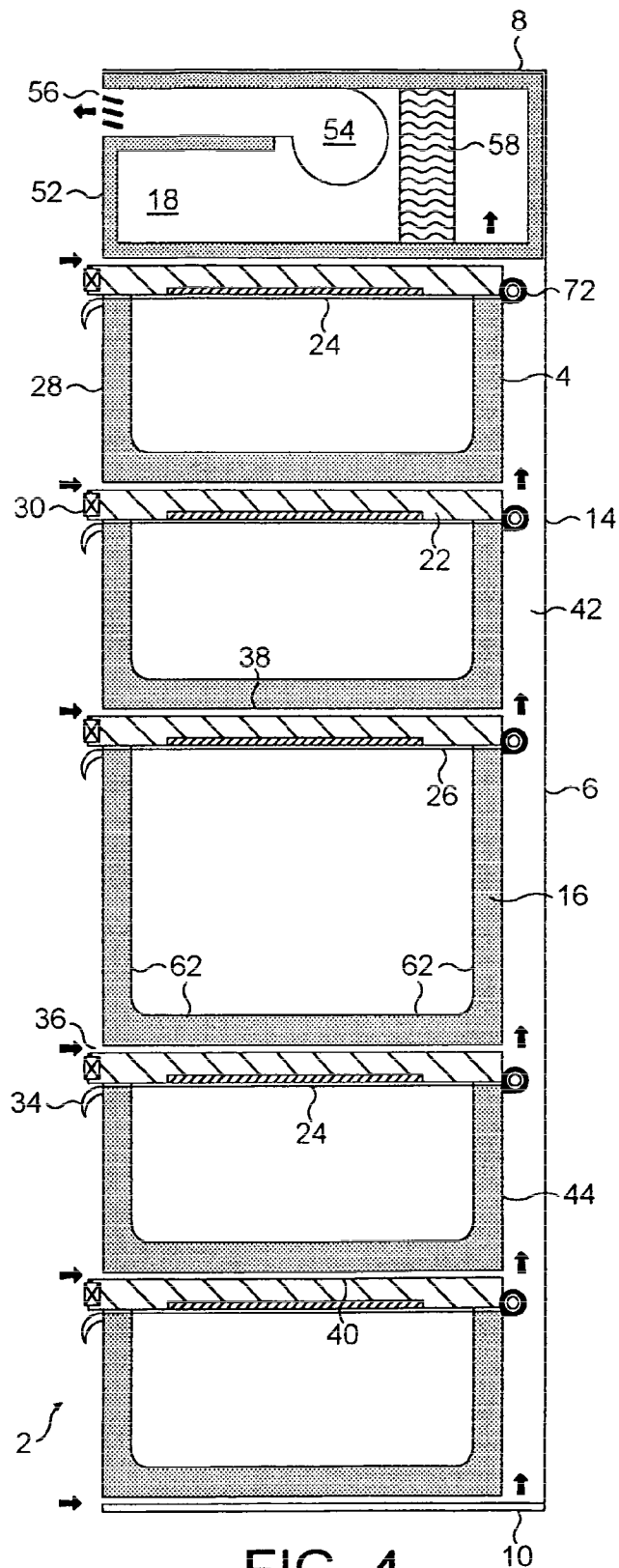


FIG. 3





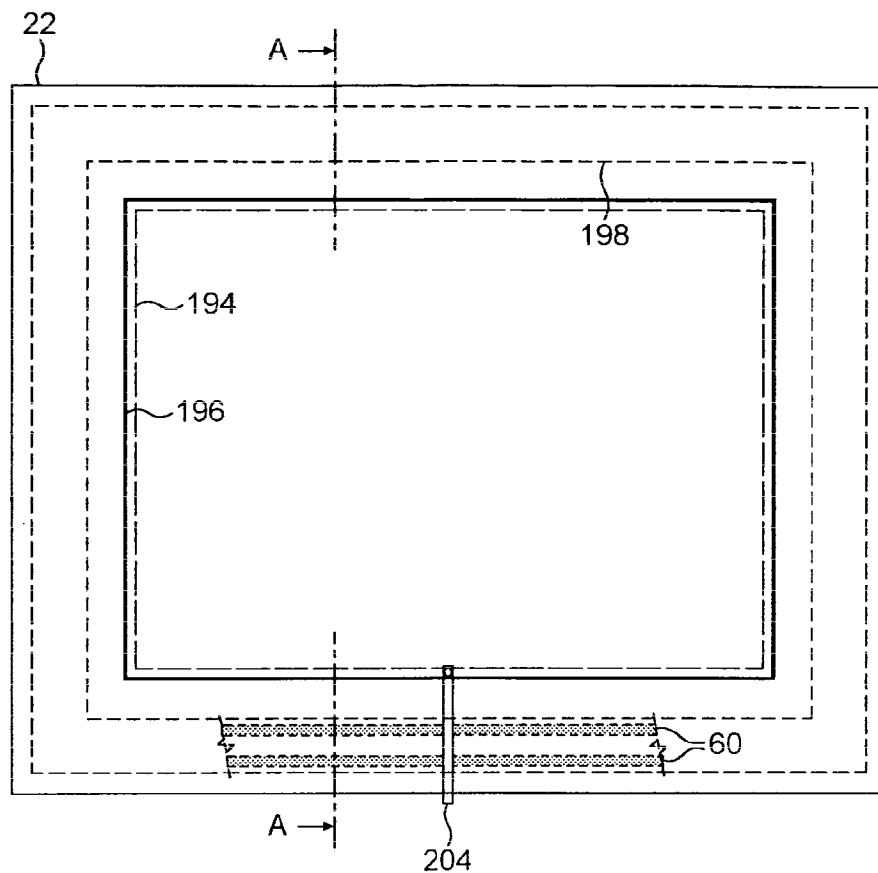


FIG. 5(a)

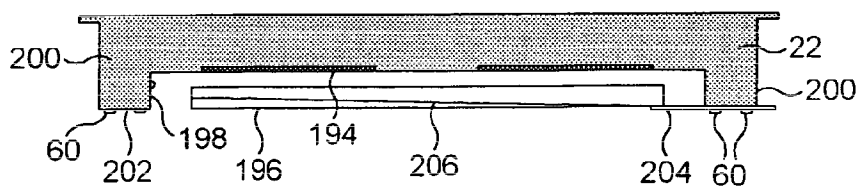
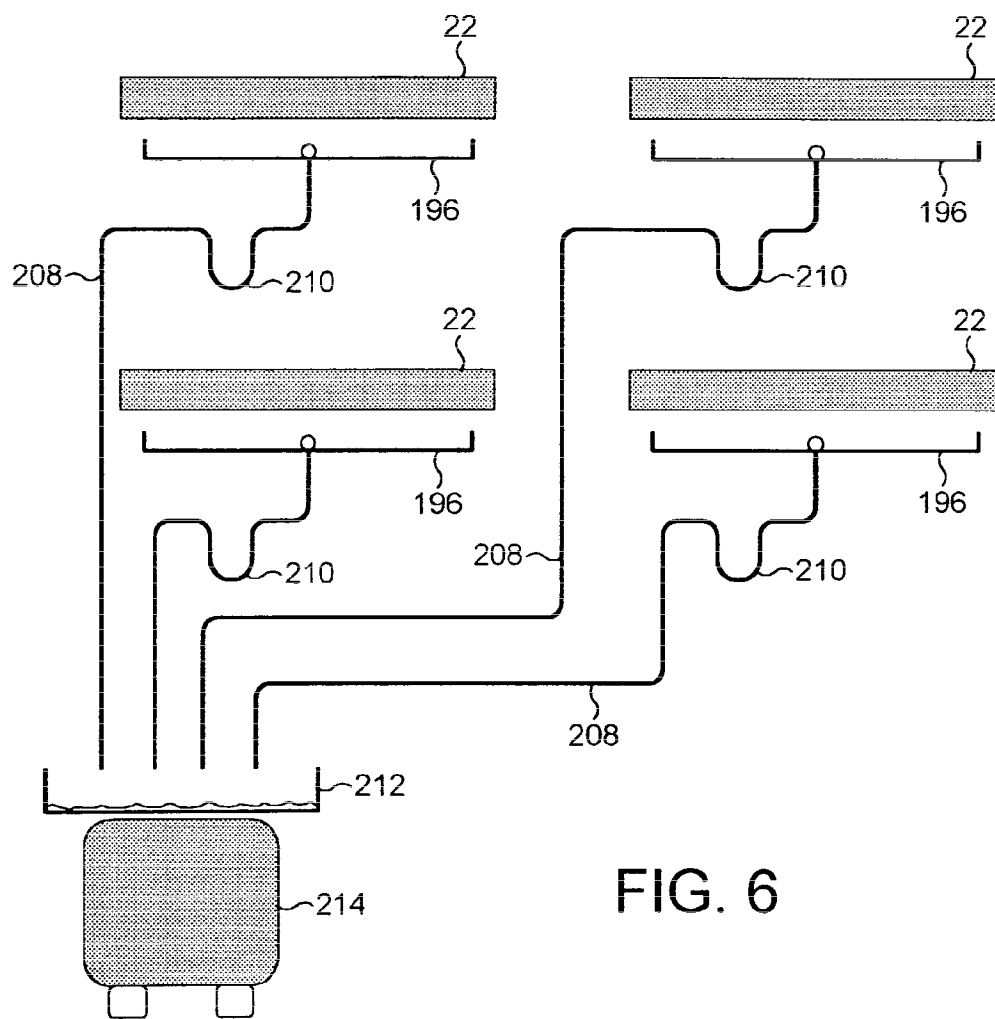


FIG. 5(b)



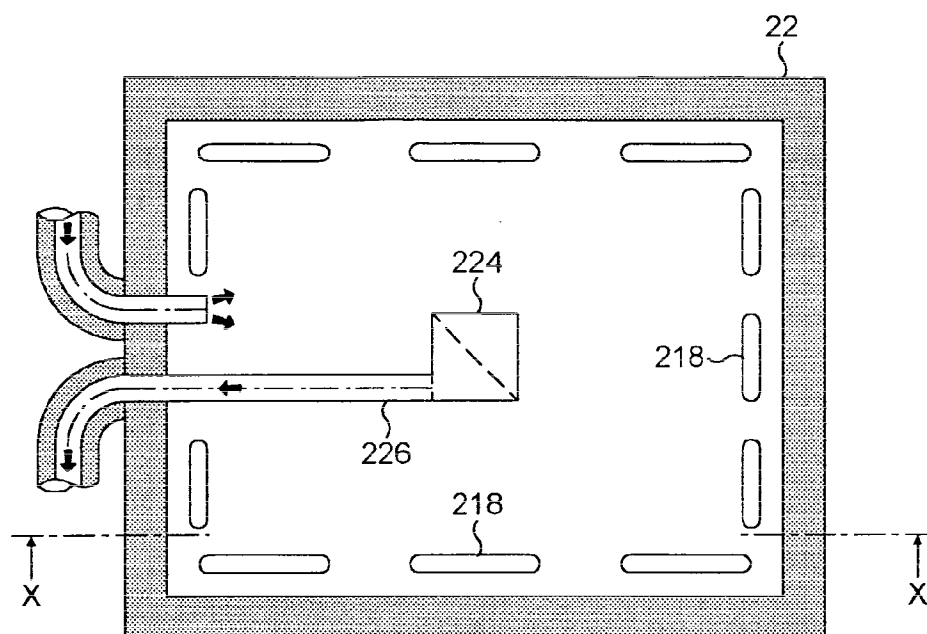


FIG. 7(a)

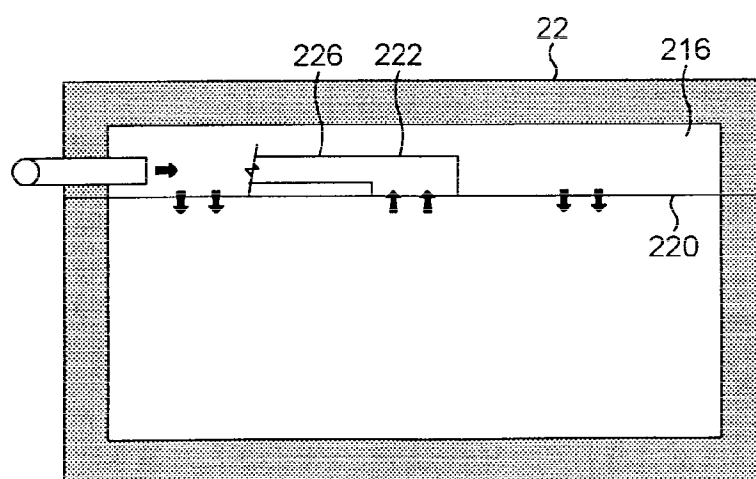
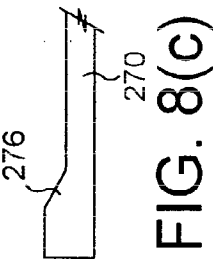
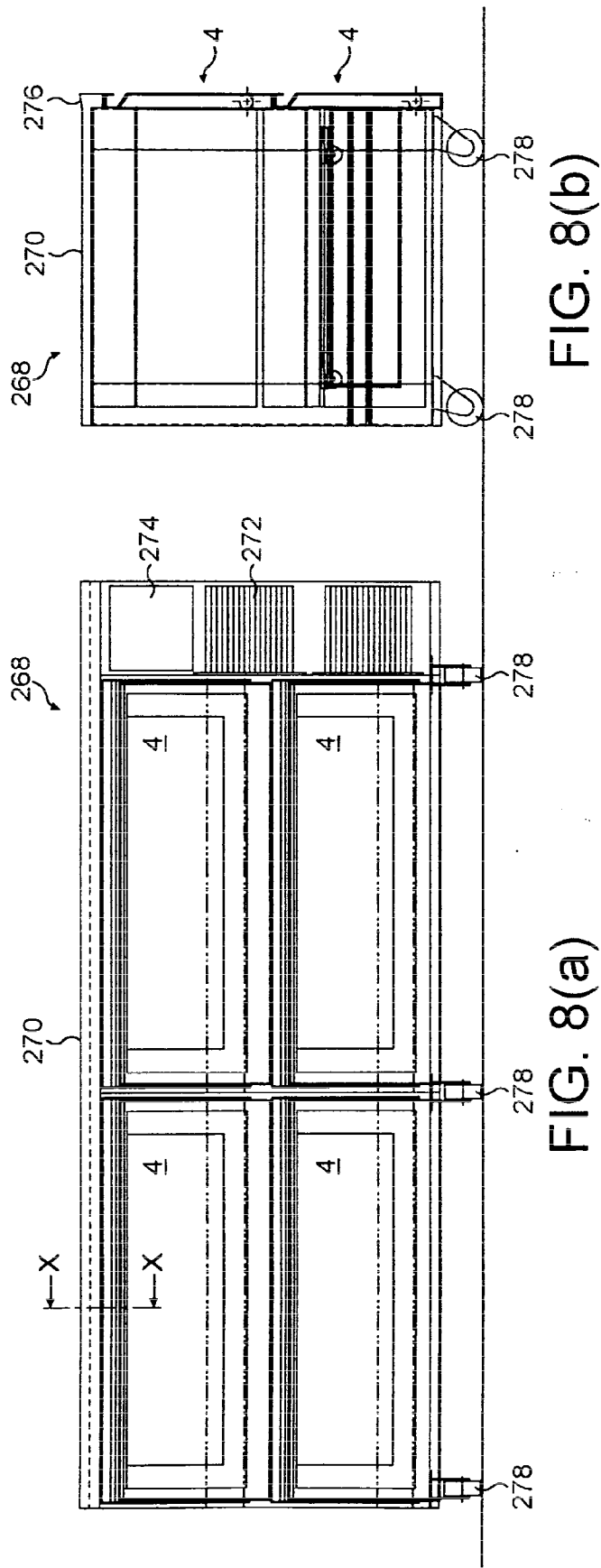


FIG. 7(b)



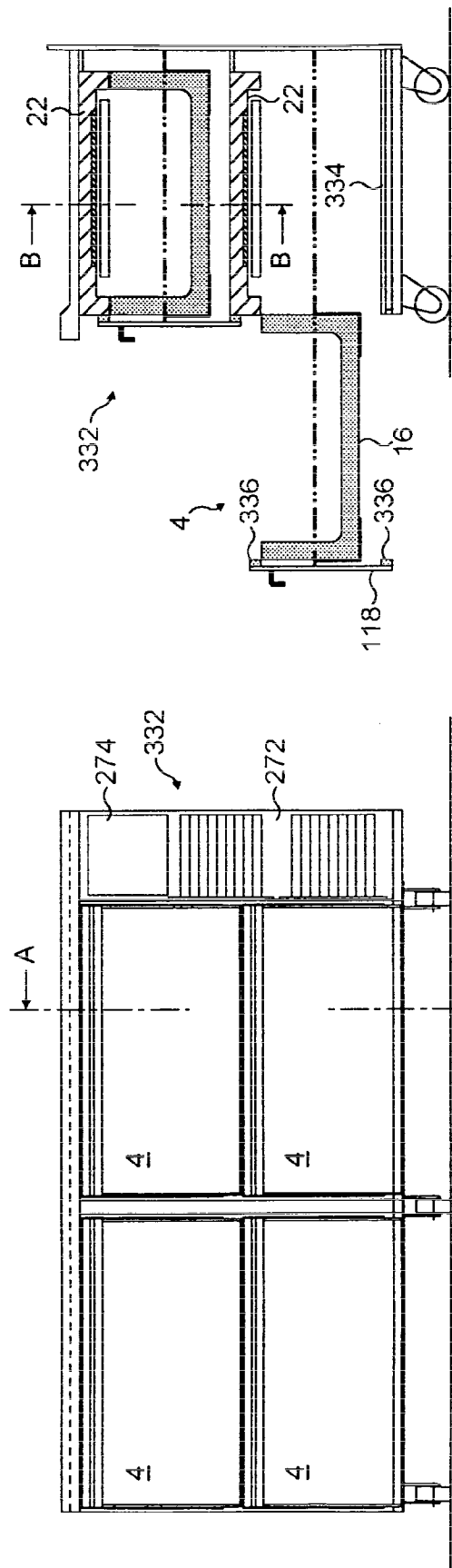


FIG. 9(b)

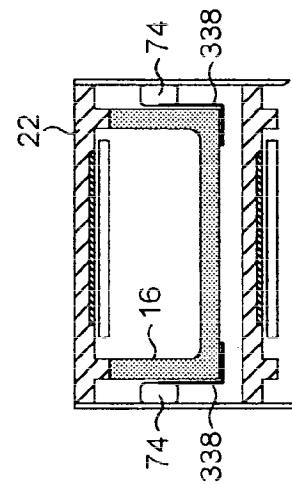


FIG. 9(c)

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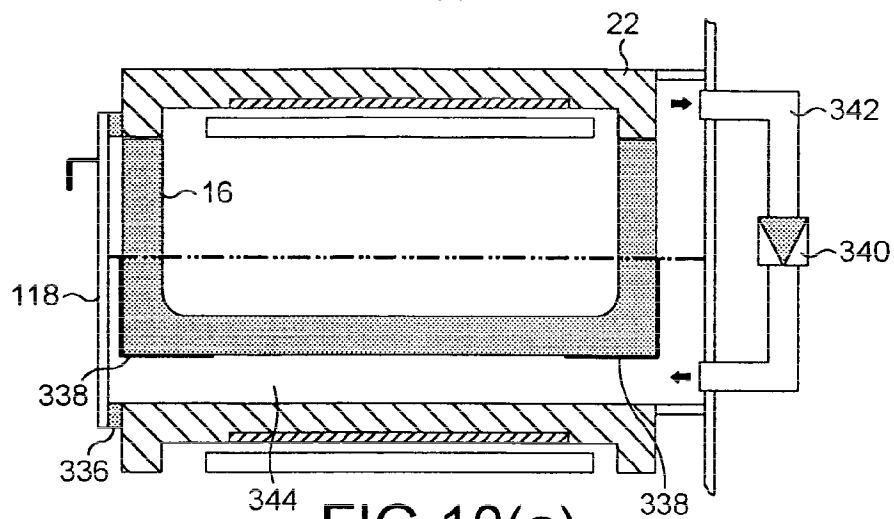


FIG. 10(a)

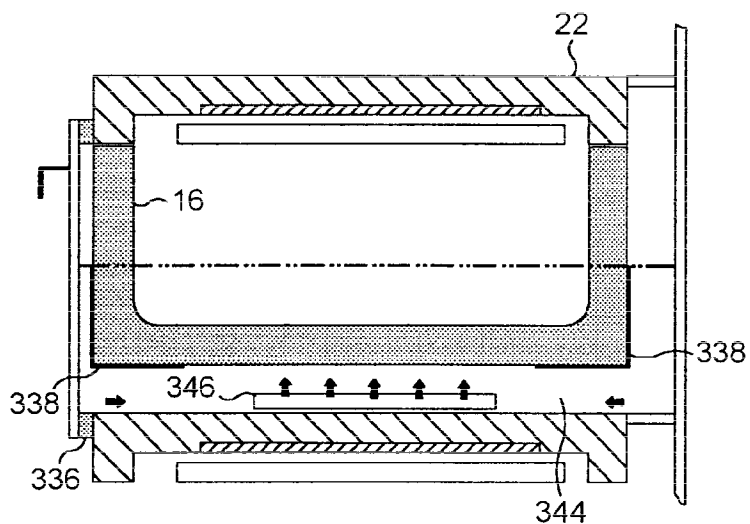


FIG. 10(b)

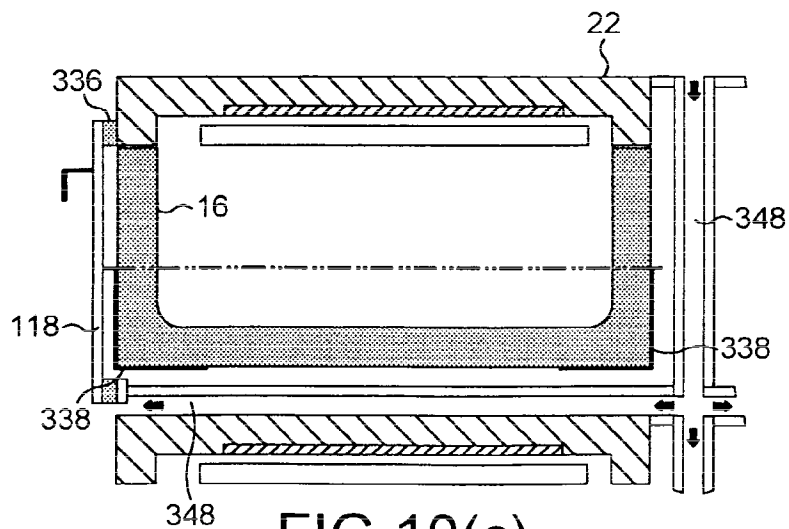


FIG. 10(c)

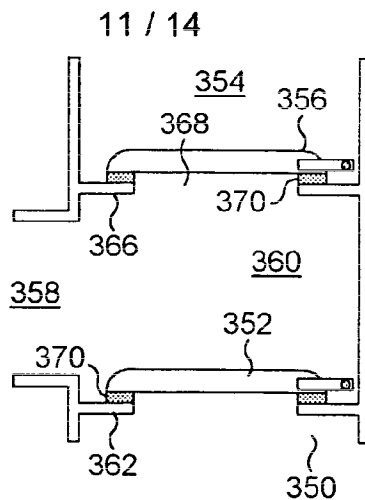


FIG. 11(a)

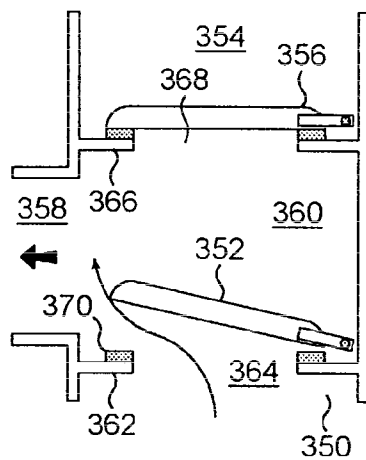


FIG. 11(b)

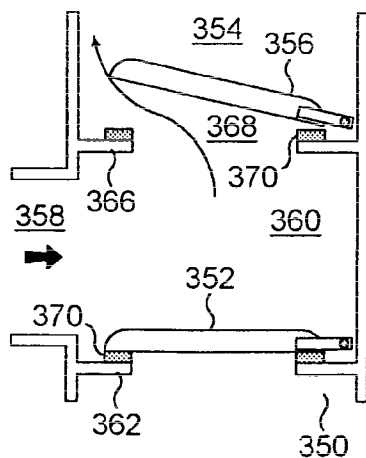


FIG. 11(c)



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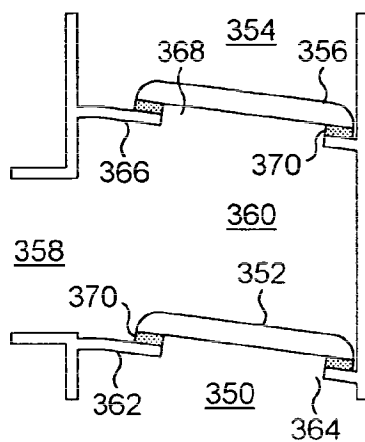


FIG. 12(a)

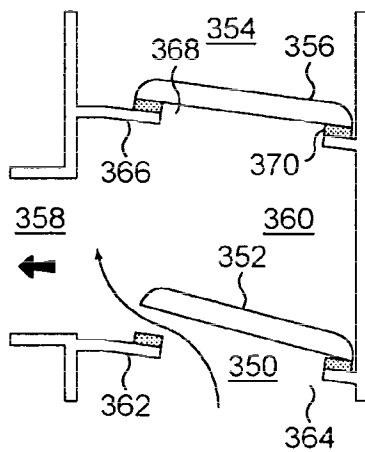


FIG. 12(b)

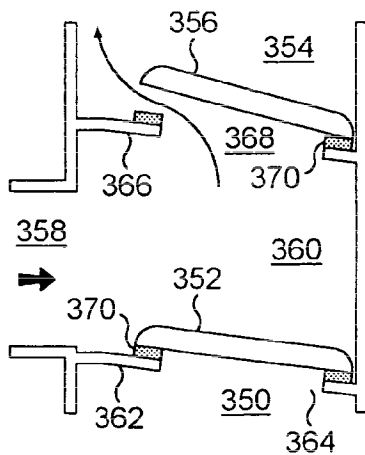


FIG. 12(c)

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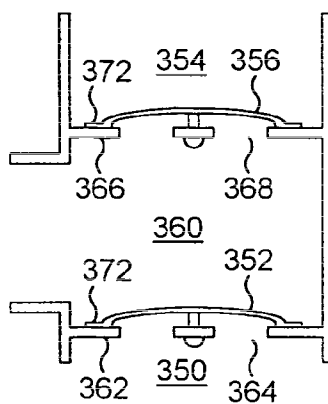


FIG. 13(a)

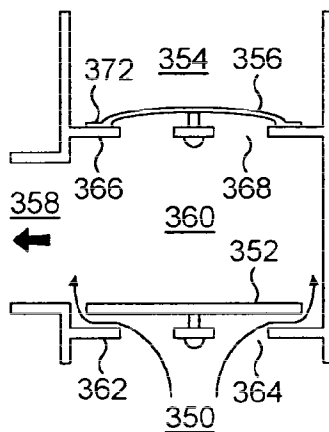


FIG. 13(b)

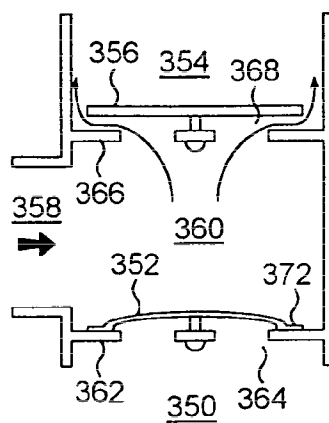


FIG. 13(c)

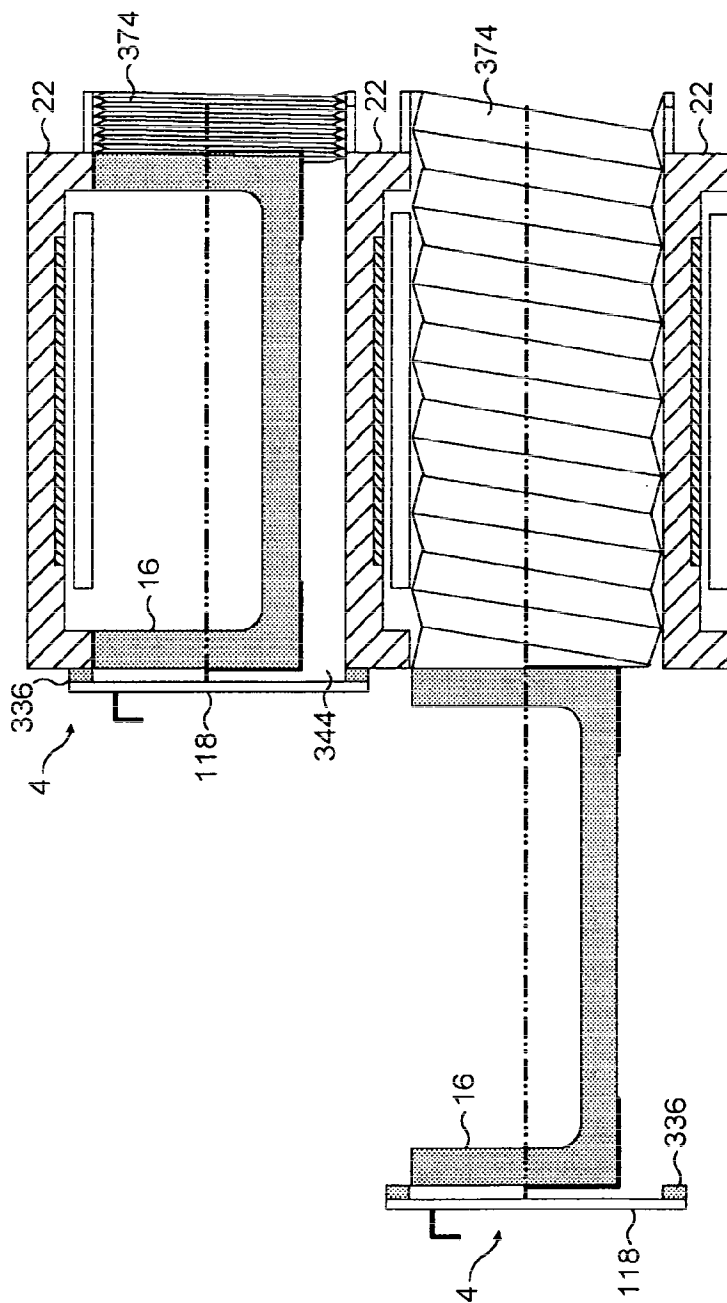


FIG. 14

## INTERNATIONAL SEARCH REPORT

Intern I Application No  
PCT/GB 02/01158

## A. CLASSIFICATION OF SUBJECT MATTER

IPC 7 F25D17/04 F25D21/04 F25D23/00 F25D25/02 F25D25/00

According to International Patent Classification (IPC) or to both national classification and IPC

## B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

IPC 7 F25D B65D B65G A47B

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal, WPI Data, PAJ

## C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category °	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
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☒ Further documents are listed in the continuation of box C.

☒ Patent family members are listed in annex.

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Date of the actual completion of the international search

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## INTERNATIONAL SEARCH REPORT

Intern: Application No  
PCT/GB 02/01158

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